

1 **CFD modeling of jet mixing to study the mixing performance and sludge Prevention in**
2 **crude oil tanks**

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9

10 **Abstract**

11 Computational fluid dynamics (CFD) model was developed using FLUENT 18.1 to study the
12 hydrodynamics in jet mixer for the prevention of sludge in a large storage tank containing crude
13 oil. Eulerian approach was used to predict the flow behavior of sludge and k-e model also used to
14 analyze the turbulence in the storage tank. The parameters such as velocity, time, nozzle head in
15 single and time, nozzle head angle position in multi jet were analyzed. The velocity at 30ms^{-1} ,
16 the mixing pattern was appreciably good and the profile obtained was fully developed at the
17 bottom. The profile developed in multi jet with various angle position also verified. RANS
18 formulation for numerical analysis also verified in which the amount at all points were not
19 exceeding 300 which was appreciable in modeling.

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21 Keywords: Jet mixing, Eulerian model, Fluent, k-e model, Hydrodynamics

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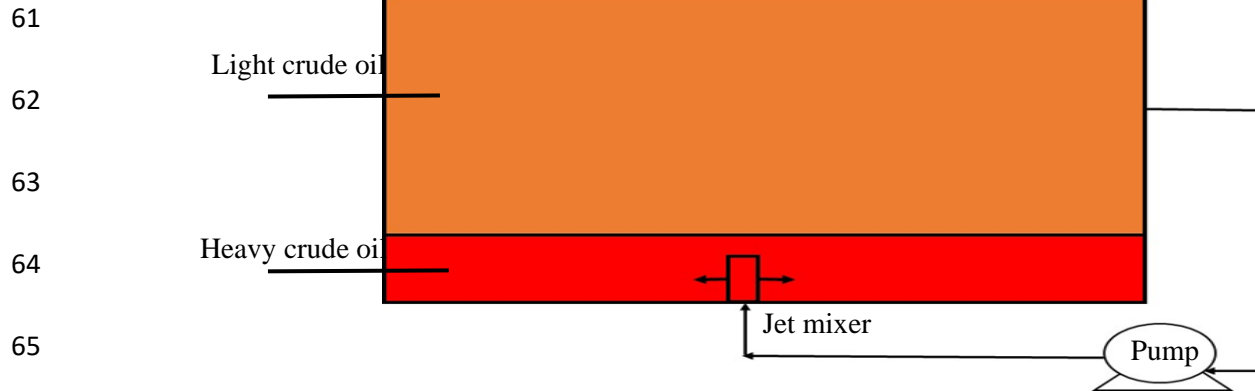
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31 **Introduction**

32 Mixing is the most general unit operations available in chemical industries. Jet mixers are
33 nowadays used instead of impellers because of its high velocity circulation without any moving
34 parts, less energy and maintenance cost, occurrence of less dead zones in larger tanks, higher
35 turbulence and shear rate, vortex motion, contact surface reduction and less in mixing time
36 (Lotfi al., 2017, Toru et al., 2012). Jet mixers are nowadays used in heavy oil storage tanks to
37 avoid the sludge formation. In jet mixers, circulatory pattern was formed leads to cause the oil
38 and sludge movement from the bottom to the middle resulting in mixing of two fluids (Paul et
39 al., 2004). There are so many parameters influencing the mixing in jet mixers like mixing time,
40 fluid density, diameter of the tank, diameter, position, nozzle diameter, geometry & velocity of jet
41 and Reynolds number (Lotfi Neyestanaket al., 2017, Raja et al., 2007).

42 Researchers have proposed various methods with various parameters to study the multiphase
43 flow in storage tanks. Numerous studies have been conducted to correlate the mixing time,
44 diameter of tank, jet velocity. On contrary, mixing time was dependant on lower Reynolds
45 number and independent of higher Reynolds number (Fosset, 1951, Fox and Gex, 1956). And
46 also, mixing time was reduced in stronger jet mounted in entry of the system (Coldrey et
47 al., 1978). Meanwhile, dissipation rate of energy governed the mixing time in a expanse long
48 away from the nozzle (Grenville and Tilton, 1996). The orientation of the jet influenced the flow
49 pattern and mixing time was dependant on diameter of nozzle and angle of inclination
50 (Patwardhan, 2002). The mixing time is an essential parameter to predict accurately for an
51 excellent design and process operation of large storage tanks. But, it needs some detailed
52 information about flow properties and hydrodynamics. In such cases, flow velocity, jet
53 parameters and mixing optimizations are hardly to attain in 2D and 3D patterns.

54 Computation Fluid Dynamics (CFD) is a forceful tool to predict all the mentioned contrary
55 parameters precisely for an effective design of storage tanks. And also, it is a supplement to the
56 lab scale data with an extensive methods to get high resolution in mixing time and fluctuations in
57 turbulent and laminar flow. So, our objective is to study the two-phase system and modeling of
58 submerged turbulent jet mixing in two phase with two dimensional (2D) and three dimensional
59 (3D) system with single and four nozzles as shown in figure 1 by CFD Eulerian approach using
60 FLUENT 18.1 to predict the mixing time in crude oil storage tank.



67 **Figure 1. Schematic representation of crude oil storage tank**

68 2. CFD Modeling

69 The geometry and mesh generated in ANSYS is shown in figure 2. The developed system is an
70 oil storage tank with dimensions of $114 \text{ m} \times 7 \text{ m}$ which is floated as a ceiling and contains sludge
71 with one meter height from tank bottom and the rest is crude oil. The nozzle, placed at 80 cm
72 from tank bottom with the diameter of 100 mm, and receives crude oil from upper layer and jets
73 it into the sludge (LoftiNeyestanak et al., 2017). The assumptions made for modeling was
74 incompressible flow, 2-dimensional flow, uniform dispersion and isothermal conditions.

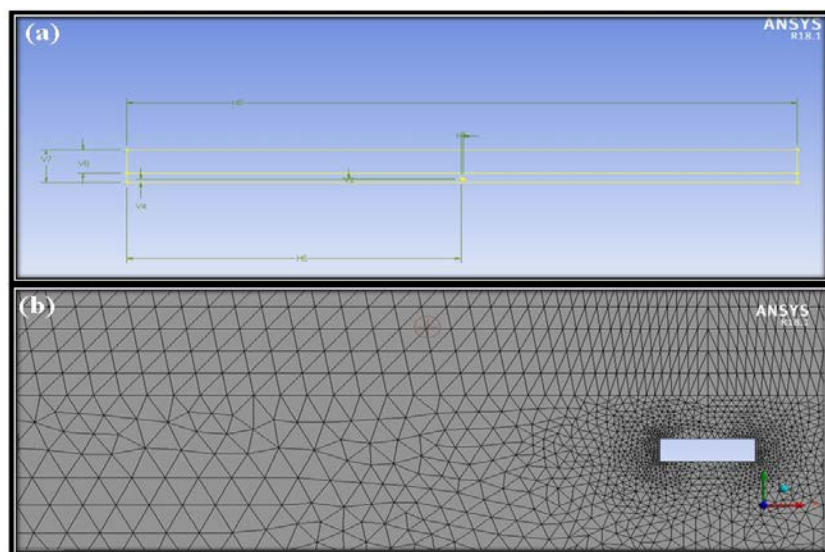
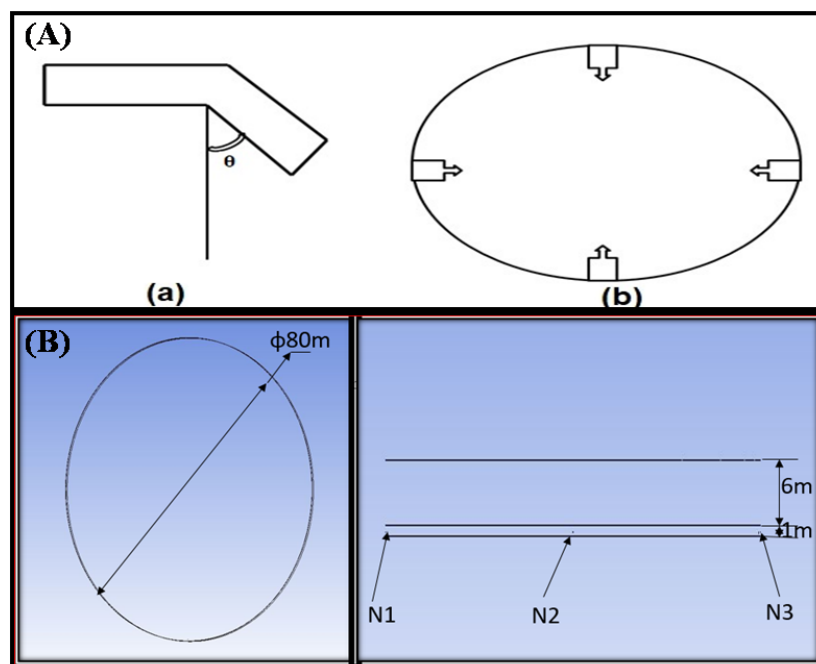


Figure 2. (a) Geometry and (b) Mesh generated by ANSYS

77 The jet mixing tank dimensions of $80\text{ m} \times 7\text{ m}$ with four jet nozzles placed at the circumference
 78 of the base of the cylindrical tank which is placed equidistant from each other. Bottom part of the
 79 tank contains one-meter sludge and upper part is the six-meter crude oil where jet is placed
 80 inside the tank wall. The nozzle is placed at various height from tank bottom which has a
 81 diameter of 100 mm , and receives crude oil from upper layer and jets it into the sludge. The four
 82 nozzles were placed at opposite to each other. The jet position and crude oil properties are shown
 83 in the table 1 and 2.

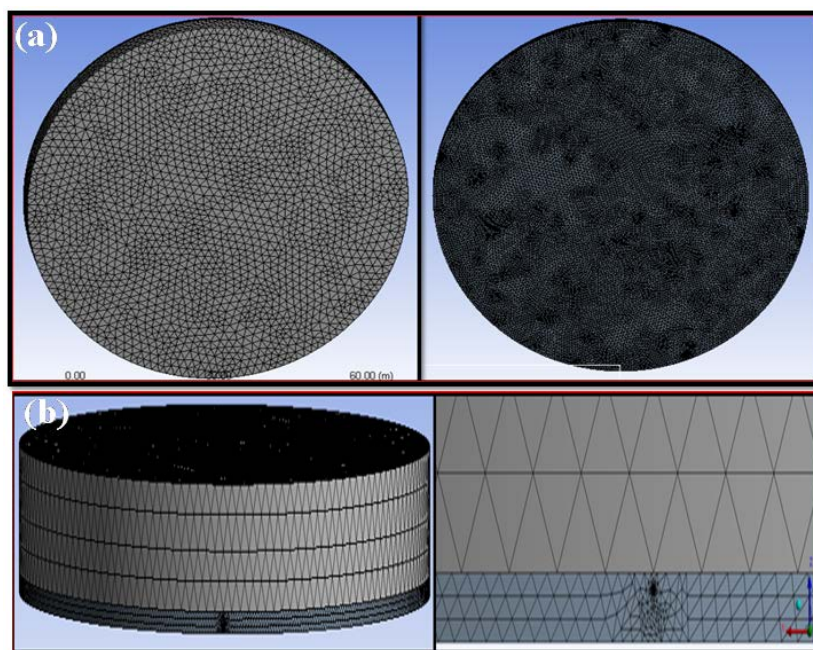


84
 85 **Figure 3. (A) Schematic representation of Jet angle with (a) 45° downwards (b) Elevation of**
 86 **four jet nozzle and (B) Geometry of the jet mixer in storage tank**

87 **Table 1. Position of jet nozzle at different height and jet angle**

Nozzle height (m)	Angle		
0			
0.3	0°	0° & 45°	45°
0.5			
0.7			

88



89

90 **Figure 4. (a) Top and bottom part mesh of the jet mixer and (b) Side view of the jet mixer**

91 It is evident from figure 4 that irregular meshing is given for the sludge part and regular meshing
 92 is given for the oil part. The purpose of meshing is to separate the domain into a convenient
 93 number location for obtaining accurate results. Grid or Mesh is defined as smaller shapes formed
 94 after discretization of geometric domain. Mesh or Grid can be in 3- dimension and 2-
 95 dimension. The mesh of crude oil part is a regular triangle type and which have the lower
 96 geometrical flexibility but faster, and the mesh of nozzle and sludge part is an irregular triangle
 97 type and which have the higher geometrical flexibility but the methods were slower.

98 The equations chosen for determining the flow pattern in multiphase flow are

99 **Continuity equation for phase q**

$$100 \quad \frac{d}{dt} \alpha_q \rho_q + \nabla \alpha_q \rho_q \vec{v}_q = \sum (\dot{m}_{pq} - \dot{m}_{qp}) + S_q \quad 1$$

101 where, the \dot{m}_{pq} represents the mass transport phase p to q, \vec{v} represents the phase velocity in ms^{-1} ,
 102 α_q represents the volume of fluid in m^3 , ρ_q is the phase density kg m^{-3} and S_q is the mass source.

103

104

105 **Momentum equation for phase q**

$$106 \frac{d}{dt}(\alpha_q \rho_q \vec{v}_q) + \nabla(\alpha_q \rho_q \vec{v}_q \vec{v}_q) = -\alpha_q \nabla P + \nabla \bar{\tau}_q + \alpha_q \rho_q \mathbf{g} + \sum_{p=1}^n (R_{pq} + \dot{m}_{pq} \vec{v}_{pq} - \dot{m}_{qp} \vec{v}_{qp}) + (\vec{F}_q + \vec{F}_{\text{lift},q} + \vec{F}_{\text{vm},q})$$

107 2

108 where, R_{pq} is the interphase between phase, P is the pressure (Pascal), F_q is the external force,

109 $F_{\text{lift},q}$ is the lift force in $\text{kg} \cdot \text{m} \cdot \text{s}^{-2}$, $\vec{F}_{\text{vm},q}$ is the virtual mass force $\text{kg} \cdot \text{m} \cdot \text{s}^{-2}$

110

111

Table 2. CFD model input parameter

Input Parameters	Value
Inlet crude oil viscosity	$9.75 \times 10^{-2} \text{ kg m}^{-1} \text{ s}^{-1}$
Crude oil density	$8.58 \times 10^2 \text{ kg m}^{-3}$
Heavy crude oil viscosity	$5.0 \times 10^2 \text{ kg m}^{-1} \text{ s}^{-1}$
Heavy crude oil density	$9.3 \times 10^2 \text{ kg m}^{-3}$

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113 **3. Results and discussion**

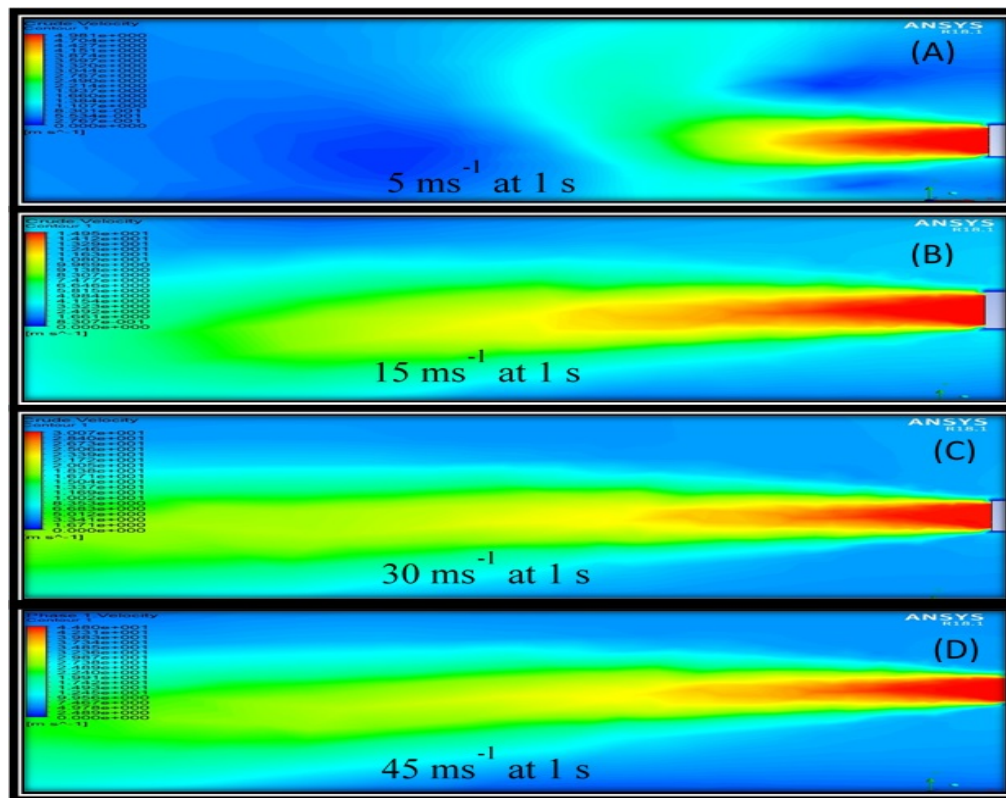
114 k-ε model has been used to find out the hydrodynamics of the fluid by first order upwind
 115 discretization and standard pressure momentum with pressure velocity algorithm. The inherent
 116 parameter considered was the jet parameters in jet flow distribution. So, CFD model was used to
 117 study the effect of velocity, time and nozzle head for single and four nozzles with
 118 various orientations.

119

120 **3.1 Analysis with single jet**121 **3.1.1 Effect of velocity on Mixing**

122 The effect of time on jet velocity was investigated. Modeling was done for mixing profile in the
 123 first second with four velocities such as 5, 15, 30 and 45 ms^{-1} . The profile was short in 5 ms^{-1} as

124 shown in figure 5 (A) because the velocity is high near the nozzle so the dissipation of fluid jet
125 into the tank contents results in diminish in velocity. The rate of penetration was moderate in 15
126 ms^{-1} as shown in figure 5 (B) which was due to the condition of no slip and force increased by the
127 fluid leads to flow path sweeping. And also, the fluid was stagnant at the surface because of the
128 lower penetration through the sludge lower layer. The velocity was further increased to 30ms^{-1} as
129 shown in figure (c), there was significant changes were observed like increase in diameter, length
130 of the penetration, radius of the jet and may reduce the sludge deposition (Randal et al., 2012).
131 This increase in all the parameters obviously increased the mixing rate from the down layer to
132 the top indicating that this velocity was optimum for mixing. The rate of mixing and depth of the
133 penetration was gradually increased at 45ms^{-1} , but it was not much effective. This can be
134 improved by increasing the mixing time (Sundarara and Selladurai, 2013).



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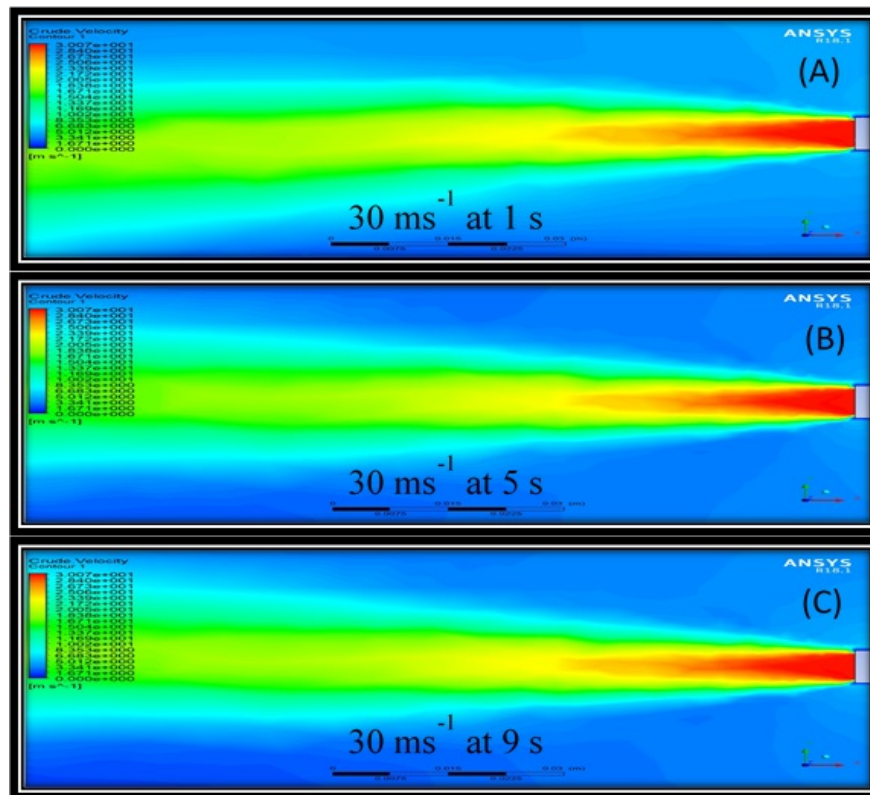
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Figure 5. Velocity distribution profile of crude oil in 1s

137

3.1.2 Effect of time on mixing

138 The velocity profile was enhanced with increased time from 1 to 9 s. As the time increased, the
139 disturbance was created the interface and proceeds to remove the stratified layer. The velocity
140 rate increased show that increased in penetration of liquid with high circulation and rotational
141 flow which leads to increase the mixing quality. The velocity profile is obvious in figure 6 (B &
142 C), the penetration length of the jet increased with reduced width over time. As the time
143 increased from 5 to 9s, the oil in the interface is forced back down the wall results in entrainment
144 (Kevin and Jeffrey, 2010). Hence, 5s was an optimum time for mixing.



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Figure 6 Jet velocity profile time (A) 1s, (B) 5s and (C) 9s

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3.1.3 Effect of nozzle head on mixing

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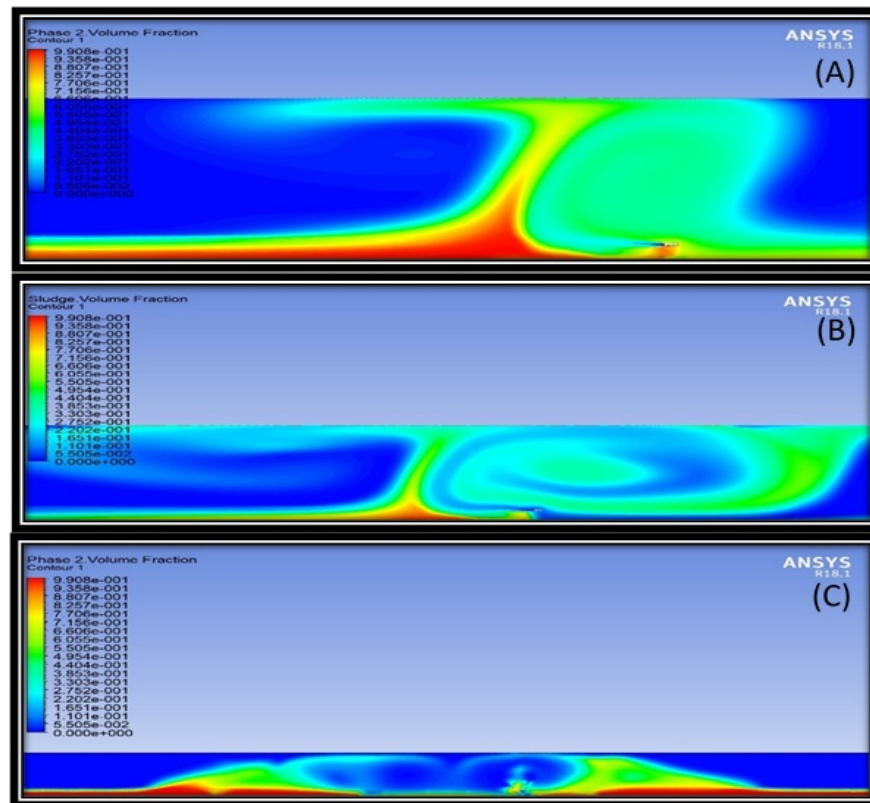
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The mixing profile distribution for the sludge at 1 m height with mixing time of 5 s was observed at velocity profile of 30ms^{-1} as shown in figure 7. The mixing was effective at 60cm as shown in figure 7 (A), the sludge in the bottom was easily mixed rather than at the top due to due to the higher-pressure head and opposing/resisting force when compared to the lift force exerted by the jet. And also, the jet dispersed the fluid radially outward the bottom with reduce in

153 velocity (Patwardhan and Gaikward, 2003).The height was further increased to 80 cm, the
154 mixing was moderate with good profile because of decrease in axial velocity by the surrounding
155 liquid entrainment. The sludge distribution profile for 5s with a jet nozzle height of 100cm in
156 7(C), the mixing of sludge is not an effective because the bottom layers remains undispersed and
157 stagnant. As the velocity increases, the resistance to the opposing force also increases and good
158 mixing rate was attained. Hence, the sludge height of 1 m and the nozzle height of 80 cm was
159 optimum for good mixing (Paul et al., 2004).



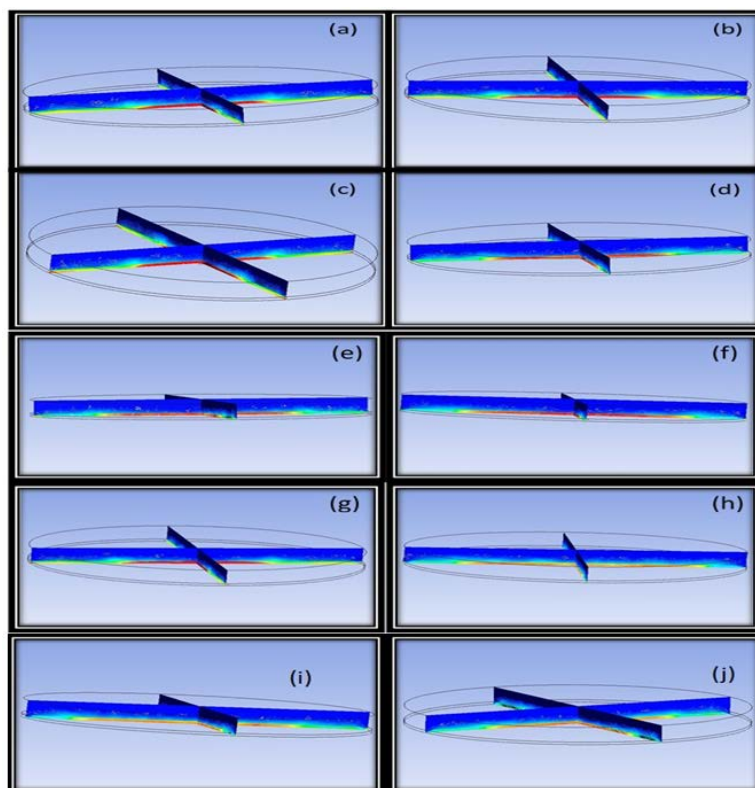
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161 **Figure7. Mixing profile distribution at jet nozzle height of (A) 60, (B) 80 and (C) 100cm**

162 3.2 Analysis with multijet

163 The multi-jet performance was observed by optimizing such variables jet location, jet angle and
164 time. Increase in time at constant velocity, causes the increase in jet path length and penetration
165 length results in a higher mixing rate and better de-sludge process. The mixing effect is
166 determined by taking average values of volume fraction of sludge at different location in the axis
167 plane. The overall average volume fraction (OAVF) of the tank was in the range of 0.804 – 0.751

168 at 30s. The OAVF value was increased at the nozzle height of 0.5 and 0.7 than 0.3 due to the
 169 improper mixing results in desludge process. However, the OAVF of overall tank was in the
 170 range of 0.804-0.751 is overrated value for the mixing process. So, the procedure was continued
 171 for 60, 90 and 150s.

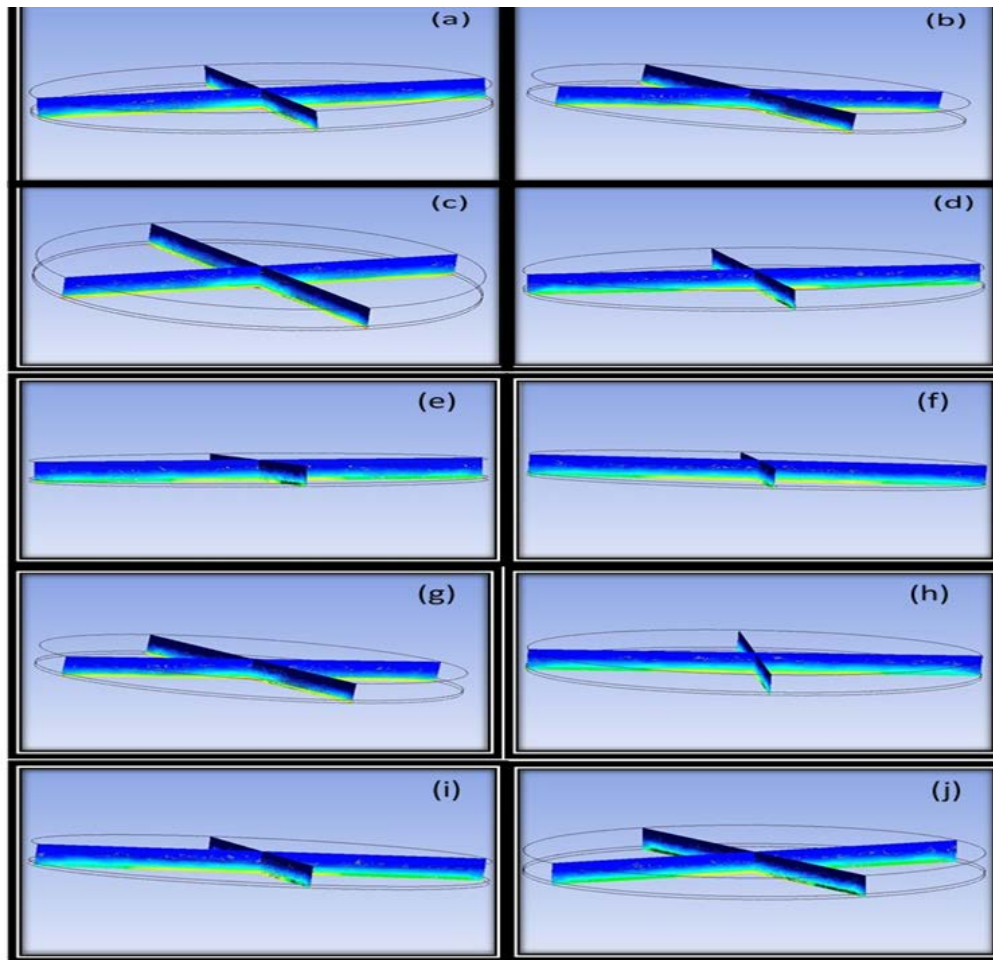


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173 **Figure 8. Sludge mixing distribution profile at 30s (A) jet height of 0.3 m, nozzles 0° , (B) jet**
 174 **height at 0.5 m nozzles 0° (C) jet height at 0.7 m nozzles 0° (D) jet height at 0.3 m with 2**
 175 **nozzles 0° and 2 nozzles 45° declined jet. (e) jet height at 0.5 m with 2 nozzles 0° and 2**
 176 **nozzles 45° declined jet., (f) jet height at 0.7 m with 2 nozzles 0° and 2 nozzles 45° declined**
 177 **jet. (g) jet height at 0.3 m Four nozzles with 45° angle downwards (h) jet height at 0.5 m**
 178 **four nozzles with 45° angle downwards. (i) jet height at 0.7 m four 45° declined jet., (j) jet**
 179 **height at 0 m four nozzles with 45° angle upwards.**

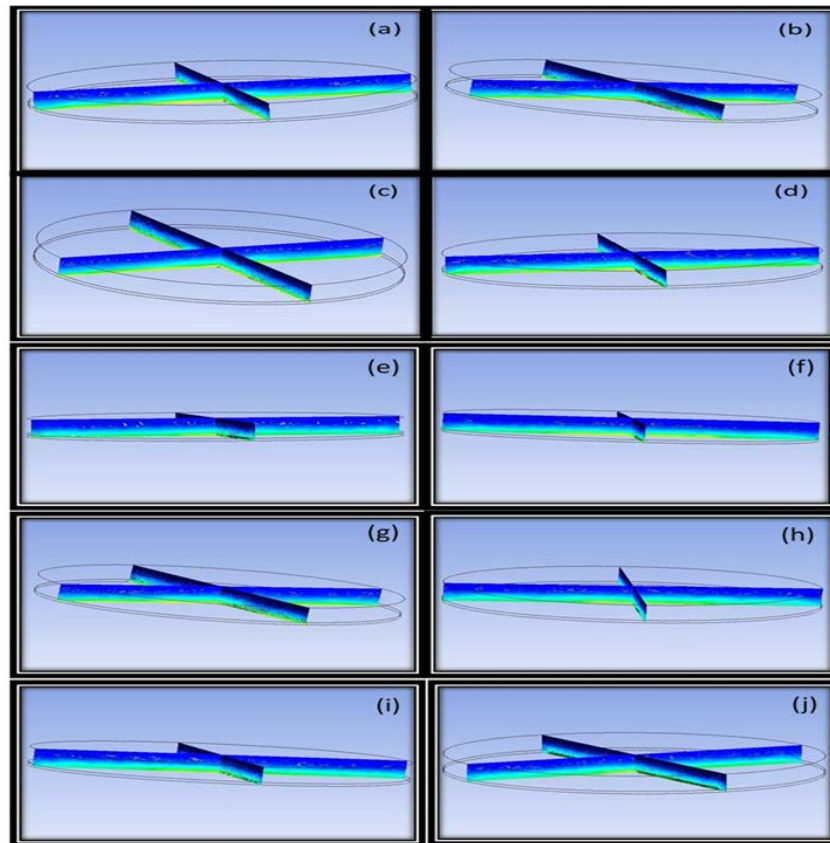
180 3.2.1 Effect of multi nozzle position at 90 s

181 The OAVF inside the tank was observed in the range of 0.715-0.635, when the time was
 182 increased from 30 to 60s. It was quite higher. So, the time was further increased to 90s, the
 183 OAVF value was decreased to 0.571 by the effect of velocity and time.



184

185 **Figure 9. Sludge mixing distribution profile at 90s (A) jet height of 0.3 m, nozzles 0° , (B)**
 186 **jet height at 0.5 m nozzles 0° (C) jet height at 0.7 m nozzles 0° (D) jet height at 0.3 m with 2**
 187 **nozzles 0° and 2 nozzles 45° declined jet. (e) jet height at 0.5 m with 2 nozzles 0° and 2**
 188 **nozzles 45° declined jet., (f) jet height at 0.7 m with 2 nozzles 0° and 2 nozzles 45° declined**
 189 **jet. (g) jet height at 0.3 m Four nozzles with 45° angle downwards (h) jet height at 0.5 m**
 190 **four nozzles with 45° angle downwards. (i) jet height at 0.7 m four 45° declined jet., (j) jet**
 191 **height at 0 m four nozzles with 45° angle upwards.**



192

193 **Figure 10. Sludge mixing distribution profile at 150s (A) jet height of 0.3 m, nozzles 0° , (B)**
 194 **jet height at 0.5 m nozzles 0° (C) jet height at 0.7 m nozzles 0° (D) jet height at 0.3 m with 2**
 195 **nozzles 0° and 2 nozzles 45° declined jet. (e) jet height at 0.5 m with 2 nozzles 0° and 2**
 196 **nozzles 45° declined jet., (f) jet height at 0.7 m with 2 nozzles 0° and 2 nozzles 45° declined**
 197 **jet. (g) jet height at 0.3 m four nozzles with 45° angle downwards (h) jet height at 0.5 m**
 198 **four nozzles with 45° angle downwards. (i) jet height at 0.7 m four 45° declined jet., (j) jet**
 199 **height at 0 m Four nozzles with 45° angle upwards.**

200

201 The mixing time plotted against the jet injected at a 20° angle with five different angle of
 202 twisting. Normally both the jets are kept at 20° with 0° twisting it will be shown in the fig 2 with
 203 the angle of twisting increases, time required for mixing increases up to 45° twisting, surprisingly
 204 the mixing time get decreased at higher angles, and at 90° twisting gave a shortest mixing time
 205 compared with the other four angle of the twisting. This procedure was repeated for three
 206 different Reynolds numbers. Reason for change in mixing time is due to change in the flow
 207 pattern of the fluid in the tank. At 90° twisting the flow pattern was looking like a circular

208 flow due to this maximum jet length has been obtained when compared to the other twisting
 209 degrees (Abdullah et al., 2017).

210

211 3.3 Effect of time on OAVF

212 The effect of time with respect to nozzle position is an important factor the OAVF was reduced
 213 when the time was increased from 30 to 150s, in which 150s of 0.3 m height nozzle has achieved
 214 the least OAVF value. The values obtained for OAVF with respect to time various nozzle height
 215 is shown in table.

216

Table 3 Effect of time and nozzle height on OAVF

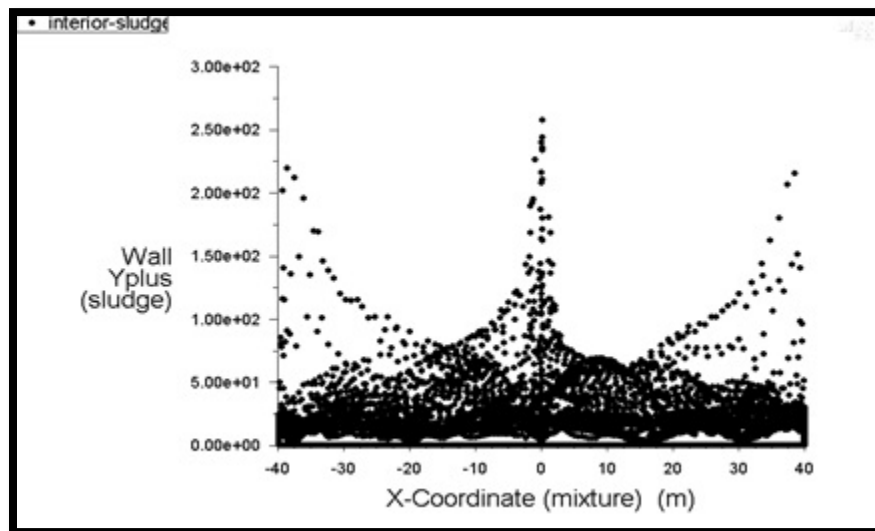
	Time (s)	Height (m)	Angle (°)		
			0	0 & 45	45
30		0.3	0.804	0.734	0.788
		0.5	0.825	0.783	0.803
		0.7	0.834	0.797	0.811
		0	-	-	0.751
60		0.3	0.715	0.668	0.67
		0.5	0.729	0.658	0.671
		0.7	0.741	0.697	0.684
		0	-	-	0.635
90		0.3	0.621	0.589	0.564
		0.5	0.617	0.583	0.589
		0.7	0.639	0.571	0.593
		0	-	-	0.591
120		0.3	0.552	0.528	0.536
		0.5	0.562	0.538	0.513
		0.7	0.584	0.528	0.523
		0	-	-	0.534
150		0.3	0.471	0.488	0.491

0.5	0.478	0.484	0.48
0.7	0.483	0.497	0.484
0	-	-	0.485

217

218 3.4 Evaluation and Verification of the Results

219 The accuracy of the results was analyzed and dependant on the turbulence models in all the
 220 numerical using RANS formulation. The wall y^+ data is a non-dimensional number similar to
 221 local Reynolds number, determining whether the influences in the wall adjacent cells are laminar
 222 or turbulent.



223

224

Figure 11. y^+ diagram of down wall

225

226 y^+ data shows sludge at the bottom and particle showing proper mixing distribution which were
 227 assessed to confirm the results. Less acceptable quantities of $y^+ > 250$ (< 300) indicates the sludge
 228 at the wall and at the bottom of the tank. The less in % deviation shows the good approximation
 229 of results obtained by modeling at time 150s in single jet position of 0.471 and multijet position
 230 of 0 and 45° as shown in table 4.

231

232

Table 4. Calculation of % Deviation

Time (s)	Volume Fraction at 0.3 m Jet Height		% Deviation
	Nozzle angle 0	Nozzle angle 0& 45	
30	0.804	0.734	8
60	0.715	0.668	6
90	0.621	0.589	5
120	0.552	0.528	4
150	0.471	0.488	3

233

234 4. Conclusion

235 A comprehensive 2-D and 3-D model for jet mixing was developed and studied using ANSYS
 236 software. In this work the sludge prevention in a crude-oil storage tank of 1 m sludge height was
 237 analysed. The effect of floating jet velocity in sludge prevention was studied using Euler-Euler
 238 method for the two-dimensional and three-dimensional CFD model crude-oil storage tank and
 239 the turbulence of the mixing flow was described using k-Epsilon model. It was observed that the
 240 increase in velocity changes the mixing pattern and improves the mixing of sludge with the crude
 241 oil. The effect of jet velocity, jet nozzle height, jet angle, number of jet and time on sludge
 242 prevention was analysed, optimum parameters were obtained from each model observations. It
 243 was predicted that effective mixing is attained when the jet velocity is at 30 m s^{-1} , jet nozzle
 244 height is 0.3 m and jet angle at 0° . From the results it was inferred that the optimum time for
 245 efficient mixing was 150 s. The results were also validated using y^+ data in which the y^+ value is
 246 below 300 is acceptable for the model. When comparing the number of jets used in the system
 247 the power required could be analysed to predict the efficiency factor.

248 Ethics approval and consent to participate

249 No animal or human material was not used

250 Consent for publication

251 There is no conflict of interest

252 Availability of data and materials

253 The corresponding author declare that the data supporting the findings of this study are available
254 within the paper and any raw data files be needed in another format they are available from the
255 corresponding author upon reasonable request.

256 Competing interests

257 The authors declare that they have no known competing financial interests or personal
258 relationships that could have appeared to influence the work reported in this paper.

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262 Author Contribution

263 Kavitha N P & Naveen kumar S: Writing – original draft, preparation and supervision.

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